

Rayleigh number was higher than the transition Rayleigh number given by Powe *et al.* [6]. However, as other work [4] suggests that this phenomenon is not two-dimensional but three-dimensional, it presumably is necessary to perform the transient 3-D analysis to simulate it.

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Deflection of a circular jet by a weak cross-flow

P. M. STEFFLER and N. RAJARATNAM

Department of Civil Engineering, University of Alberta, Edmonton, Canada T6G 2G7

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NOMENCLATURE

b	transverse length scale (also used as length scale for other directions)
f	function
g	function
I	constant
K	numerical constant
U	ambient longitudinal velocity
U_0	initial jet velocity
u	longitudinal velocity
u_m	maximum jet velocity at a section
V	ambient lateral velocity
v	lateral velocity
w	vertical velocity
x	longitudinal coordinate
y	lateral coordinate
z	vertical coordinate.

Greek symbols

α	ratio of initial jet velocity to cross-stream velocity
ζ	non-dimensional vertical coordinate
η	non-dimensional lateral coordinate
θ	momentum thickness
ρ	fluid density
τ	shear stress.

INTRODUCTION

THE PROBLEM of predicting the trajectory of a jet injected into a moving ambient fluid is of much importance and has been extensively studied both theoretically and experimentally. The authors have recently developed a simple method for the calculation of the deflection of a plane non-buoyant jet issuing

into a weak uniform cross-flow [1, 2]. The objective of this study is to extend this method to the much more complex and practical case of a circular jet in a moving ambient flow.

The calculation method is based on a similarity analysis of the axial momentum equation. The special case of the trajectory of a circular jet in a simple cross-flow will be compared with some of the available experimental results.

CIRCULAR JET IN A GENERAL CROSS-FLOW

Figure 1 shows a circular jet issuing from a nozzle of radius r_0 with a velocity of U_0 and exposed to a cross-flow of velocity V (which may be a function of x) and a uniform flow U in the axial direction. The trajectory is given by $\bar{y}(x)$. The continuity equation and the equation of motion for the deflected jet are

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (1)$$

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} = \frac{1}{\rho} \left(\frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} \right) \quad (2)$$

where u, v, w are the time-averaged velocities in the x, y and z directions and τ_{yx} and τ_{zx} are the turbulent shear stresses acting in the axial direction (the laminal stresses have been neglected). For convenience, the velocities may be considered as a sum of local and background components:

$$u = U + u' \quad (3)$$

$$v = V + v' \quad (4)$$

where the prime denotes a local component. These relations may be introduced into equations (1) and (2) with u and v now representing u' and v'

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (5)$$

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} + U \frac{\partial u}{\partial x} + V \frac{\partial u}{\partial y} = \frac{1}{\rho} \left(\frac{\partial \tau_{yx}}{\partial y} + \frac{\partial \tau_{zx}}{\partial z} \right) \quad (6)$$

Integrating equation (6) with respect to z first and then y from $-\infty$ to ∞ , one obtains the results

$$\frac{d}{dx} \iint_{-\infty}^{\infty} u(U+u) dy dz = 0. \quad (7)$$

This equation indicates that excess momentum flux is conserved in the x direction. Multiplying equation (6) by u and integrating as in the previous case

$$\frac{d}{dx} \iint_{-\infty}^{\infty} u^2(U+u) dy dz = -\frac{2}{\rho} \iint_{-\infty}^{\infty} \left(\tau_{yx} \frac{\partial u}{\partial y} + \tau_{zx} \frac{\partial u}{\partial z} \right) dy dz \quad (8)$$

This is the integral energy equation for an axisymmetric jet (written in Cartesian coordinates) with a general cross-flow. It

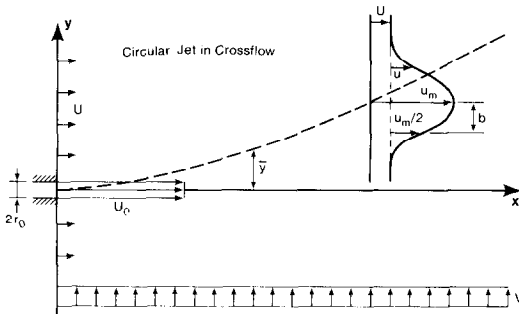


FIG. 1. Definition sketch.

is to be noted that V does not appear in equations (7) and (8). If it is assumed that the length scales are the same in both cross-sectional directions, then these equations can be solved for u_m and b , the characteristic scales.

Multiplying equation (6) by y and integrating with respect to z first and then with respect to y , from $-\infty$ and $+\infty$ (in both cases), after some simplifications, one obtains the result

$$\begin{aligned} \frac{d}{dx} \iint_{-\infty}^{\infty} yu(U+u) dy dz - \iint_{-\infty}^{\infty} uv dy dz - \iint_{-\infty}^{\infty} uV dy dz \\ = -\frac{1}{\rho} \iint_{-\infty}^{\infty} \tau_{yx} dy dz. \end{aligned} \quad (9)$$

The shear stress integral on the right-hand side of equation (9) will vanish if a velocity gradient model with a constant momentum diffusivity is assumed. In any event, this term will be small due to approximate antisymmetry of the shear stress distribution. The second term on the left-hand side may also be neglected in comparison to the third term since v is of the same order of magnitude as V (at most about 10% of initial jet velocity U_0) but is of opposite sign on either side of the jet axis while V is a constant. In addition v is smallest where u is largest. It is reasonable then to expect that the second integral will be at least one order of magnitude smaller than the third. This approximation is weakest near the jet outlet where v is largest. With these approximations equation (9) will reduce to

$$\frac{d}{dx} \iint_{-\infty}^{\infty} yu(U+u) dy dz = \iint_{-\infty}^{\infty} uV dy dz. \quad (10)$$

Equation (10) may be interpreted as saying that the rate of change of the flux of moment of momentum is equal to the convection of the u velocity in the y direction.

To proceed further, assume

$$\frac{u}{u_m} = f_1\left(\frac{z}{b}\right)g_1\left(\frac{y-\bar{y}}{b}\right) = f_1(\zeta)g_1(\eta) \quad (11)$$

$$\frac{\tau_{yx}}{\rho u_m^2} = f_2(\zeta)g_2(\eta) \quad (12)$$

$$\frac{\tau_{zx}}{\rho u_m^2} = f_3(\zeta)g_3(\eta) \quad (13)$$

wherein $\zeta = z/b$, $\eta = (y-\bar{y})/b$ and f_1, f_2, f_3, g_1, g_2 and g_3 denote functions. Substituting equations (11)–(13) into equation (10) and after some simplifications, one obtains the results

$$\begin{aligned} \frac{d}{dx} u_m b^2 \iint_{-\infty}^{\infty} f_1 g_1 \bar{y}(U+u_m f_1 g_1) d\eta d\zeta \\ = V u_m b^2 \iint_{-\infty}^{\infty} f_1 g_1 d\eta d\zeta. \end{aligned} \quad (14)$$

Since in the three final equations, equations (7), (8) and (14), equations (7) and (8) are in principle the same as those for the jet in a coflowing stream without a deflecting flow, the scales u_m and b can be taken from the study of Pande and Rajaratnam [3] as

$$\frac{U}{u_m} \approx \frac{1}{K} \left(\frac{x}{\theta} \right) \quad \text{for} \quad \frac{x}{\theta} < 150 \quad (15)$$

where $1/K$ is the slope of the curve in Fig. 6 of reference [2] and K has the approximate value of 12.2

$$b^2 = \frac{1}{2} \frac{\theta^2}{(u_m/U)^2 F_2 + (u_m/U) F_1} \quad (16)$$

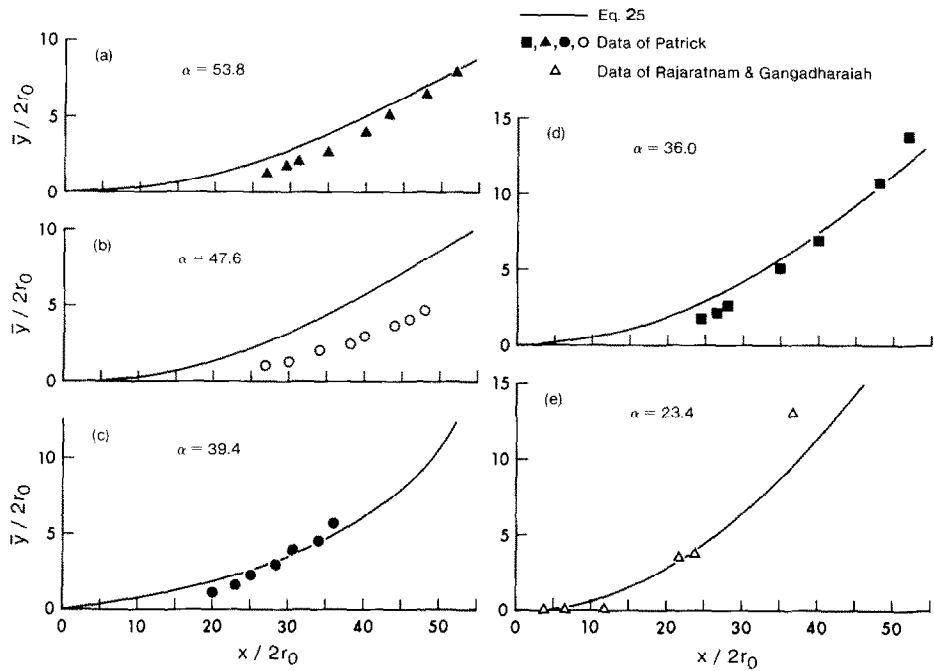


FIG. 2. Circular jets in weak cross-flow. Comparison of equation (25) with experimental results.

where θ is the momentum thickness given by the equation

$$\theta = r_0 \left[\frac{U_0}{U} \left(\frac{U_0}{U} - 1 \right) \right]^{1/2} \tag{17}$$

U_0 being the velocity of the jet at the nozzle and F_1 and F_2 are integral constants [3]. Equation (14) can be rewritten as

$$\frac{d}{dx} [\mu_m b^2 \bar{y} (U I_1 + u_2 I_2)] = V \mu_m b^2 I_1 \tag{18}$$

wherein (if exponential functions are utilized for f_1 and g_1 , [4])

$$I_1 = \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} f_1 g_1 \, d\eta \, d\zeta = \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \exp [-0.693(\eta^2 + \zeta^2)] \, d\eta \, d\zeta \tag{19}$$

$$I_2 = \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} f_1^2 g_1^2 \, d\eta \, d\zeta = \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} \exp [-1.386(\eta^2 + \zeta^2)] \, d\eta \, d\zeta. \tag{20}$$

Using equation (7), equation (18) can be reduced to

$$\frac{d\bar{y}}{dx} = \frac{I_1 V}{(I_1 U + I_2 u_m)}. \tag{21}$$

Using equations (15) and (16) and the result $I_2/I_1 = 1/2$, equation (21) can be reduced to

$$\frac{d\bar{y}}{dx} = \frac{V}{U} \frac{(x/\theta)}{[(x/\theta) + K/2]}. \tag{22}$$

For the case of a uniform cross-flow ($V = \text{constant}$) equation (22) may be integrated using $\bar{y} = 0$ at $x = 0$ to give

$$\frac{\bar{y}}{\theta} = \frac{V}{U} \left[\frac{x}{\theta} - \frac{K}{2} \ln \left\{ \frac{(x/\theta + K/2)}{K/2} \right\} \right]. \tag{23}$$

For the case of a simple circular jet in a weak uniform cross-

flow, $U = 0$ and with some algebra may be reduced to

$$\frac{d\bar{y}}{dx} = \frac{2Vx}{Kr_0 U_0}. \tag{24}$$

Integrating

$$\frac{\bar{y}}{r_0} = \frac{V}{K U_0} \left(\frac{x}{r_0} \right)^2. \tag{25}$$

Equation (25) indicates that the deflection \bar{y} is proportional to x^2 . Figure 2 shows a comparison of equation (25) with some of the (readily available) experimental observations for large values of the ratio α of the jet velocity U_0 to the cross-stream velocity V . Considering that K was evaluated from rather general considerations (i.e. observations on jets in coflowing streams) the agreement of equation (25) with the experimental results in Fig. 2 should be considered very encouraging.

CONCLUSIONS

Presented herein is a simple analytical method for calculation of trajectories of circular jets in weak cross-flows. Based on the assumption of general similarity of the jet, a simple convection relation is derived with no experimental constants to evaluate. The result is derived for an asymptotically weak cross-flow but may be applied as a first approximation to many practical problems. Since the present derivations do not assume a uniform cross-flow in the x direction, non-uniform cross-flows could also be included. The agreement of the predicted deflection of circular jets with experimental observations in weak cross-flows has been found to be satisfactory.

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A Monte Carlo procedure for straight convecting boundaries

L. C. BURMEISTER

Mechanical Engineering Department, University of Kansas, Lawrence, KS 66045, U.S.A.

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NOMENCLATURE

a coefficient
 A upper limit of integration, $A = \int_{-\pi/2}^{\pi/2} P(z) dz$

b coefficient
 B coefficient,

$$B = \int_{-\pi/2}^{\pi/2} [1 - (B_i/3) \cos(3z)] P(z) dz$$

B_i Biot number, $B_i = hR/k$

C constant

d_{ave} average distance reflected into interior perpendicular to boundary

e natural number, $e = 2.71828 \dots$

E sum of squared errors, see equation (16)

f_n function of z , $f_n(z)$

$$\begin{aligned} & \cos(nz) + B_i \cos[(n+1)z]/(n+1), n \text{ even} \\ & \sin(nz) + B_i \sin[(n+1)z]/(n+1), n \text{ odd} \end{aligned}$$

$\overline{f_n^2}$ function of $B_i, \overline{f_n^2}$

$$\begin{aligned} & \pi + 4B_i + \pi B_i^2/2, n = 0 \\ & \pi/2 + 4B_i/(2n+1) + \pi B_i^2/[2(n+1)^2], n \neq 0 \end{aligned}$$

F cumulative distribution function,

$$F = \int_{-\pi/2}^z P(z') dz'/A$$

G function of x , see equation (10)

h heat transfer coefficient

k thermal conductivity

L distance reflected in one-dimensional approximation

L_1, L_2 differential operators, see equation (10)

N random number

P probability density function, see equation (17)

q negative derivative of T with respect to z ,
 $q = -\partial T/\partial z$

q'' heat release rate per unit area

q_0 function of x , see equation (13)

Q dimensionless heat release rate parameter, Q
 $= q'' R^2/4k$

r distance from the vertex

R straight side length

T temperature

$T_{boundary}$ boundary temperature

$T_{interior}$ temperature at distance L into interior

T_0 function of x , see equation (12)

T_p peripheral temperature

T_v vertex temperature, $T_v = T(x \rightarrow \infty)$

T_∞ environmental temperature

x dimensionless coordinate, $x = \ln(R/r)$

y temperature parameter,

$$y = T_p(z) - T_\infty + Q[1 - (B_i/3) \cos(3z)]$$

z dimensionless coordinate, $z = \theta - \theta_m/2$.

Greek symbols

α differential operator, $\alpha = \partial(\)/\partial x$

θ angle from the straight side

θ_m included angle

ζ differential operator, $\zeta = \partial(\)/\partial z$

π natural number, $\pi = 3.14159 \dots$

Subscripts

m index

n index.

INTRODUCTION

THE FLOATING random walk Monte Carlo method for two-dimensional steady conduction requires a procedure to determine the next move of a walker situated on a boundary that convectively exchanges heat with an environment at known temperature across a known heat transfer coefficient. Previously [1], a one-dimensional finite difference approximation to the convective boundary condition has been employed in the direction perpendicular to the boundary as

$$T_{boundary} = [T_{interior} + (hL/k)T_\infty]/[1 + hL/k].$$

Such an approximation requires the reflective move back into the solid to be small to preserve accuracy and to be perpendicular to the boundary. A two-dimensional approximation is subject to the same difficulties.

An amelioration will be devised from a solution for the vertex temperature of a sector of a circle experiencing convection along its straight sides and with a specified temperature along the circular arc, the vertex being on the convective boundary.